# Towards Automatic Identification of Globally Valid Geometric Flat Outputs via Numerical Optimization 

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#### Abstract

Differential flatness enables efficient planning and control for underactuated robotic systems, but we lack a systematic and practical means of identifying a flat output (or determining whether one exists) for an arbitrary robotic system. In this work, we leverage recent results elucidating the role of symmetry in constructing flat outputs for free-flying robotic systems. Using the tools of Riemannian geometry, Lie group theory, and differential forms, we cast the search for a globally valid, equivariant flat output as an optimization problem. An approximate transcription of this continuum formulation to a quadratic program is performed, and its solutions for two example systems achieve precise agreement with the known closed-form flat outputs. Our results point towards a systematic, automated approach to numerically identify geometric flat outputs directly from the system model, particularly useful when complexity renders pen and paper analysis intractable.


## I. INTRODUCTION

Differential flatness is a powerful property enjoyed by many robotic systems [1], which has been employed for efficient planning and control of dynamic maneuvers [2][4], including the explicit consideration of perception and actuation constraints [5], [6]. However, the construction of a flat output for complex systems such as robots is usually accomplished by human intuition or trial and error [7], [8], since constructive approaches to flat output identification have traditionally been limited to special cases [9].

To fill this gap, several schemes for identifying an approximate numerical flat output for a given system have been proposed. The approach of [10] involves generating a dataset of state and input trajectories and representing the candidate flat output and the inverse flatness diffeomorphism as weighted sums of basis functions. However, the resulting numerical flat output is only valid locally around the sampled trajectories, the basis functions chosen seem to reflect knowledge of an analytical flat output, and the approach becomes intractible for large bases. The identification of flat outputs from data in the complete absence of an analytical system model was considered in [11], but similar locality limitations remained.

In our recent work [12], we have shown that the symmetry and Riemannian structure present in free-flying robotic systems facilitates the construction of global, equivariant, groupvalued (i.e. "geometric") flat outputs. In particular, such flat outputs can be obtained from any section of the system's principal bundle that is orthogonal to a distribution given
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in terms of the kinetic energy metric and the unactuated velocities. While this poses a geometric problem of lower dimensionality, it is not clear how to obtain closed-form solutions for such sections in general, nor are they unique, raising questions of optimality. In this work, we pursue the automatic identification of such geometric flat outputs, using finite element methods to solve the underdetermined partial differential equations arising from the orthogonality constraint. The problem bears similarities to the identification of "optimal coordinates" (i.e. an optimal trivialization) for locomotion systems [13], from which we take inspiration.

## II. Mechanical Systems with Symmetry

In this section, we give a brief review of essential concepts, and direct the reader to [14] for a comprehensive overview, as well as [12] for an introduction to differential flatness of mechanical systems with symmetry. We consider unconstrained mechanical systems with a Lagrangian of the form

$$
\begin{equation*}
L: T Q \rightarrow \mathbb{R}, v_{q} \mapsto \frac{1}{2}\left\langle\left\langle v_{q}, v_{q}\right\rangle\right\rangle-P(q) \tag{1}
\end{equation*}
$$

subject to controlled external forces contained in $F \subseteq T^{*} Q$. We assume the system has "broken" symmetry, meaning the kinetic energy metric $\langle\langle\cdot, \cdot\rangle\rangle$ and the forces $F$ are respectively invariant and equivariant with respect to the free and proper action $\Phi_{g}: Q \rightarrow Q$ of a Lie group $G$, but the potential $P$ need not be. This action also induces a projection $\pi: Q \rightarrow S$, where $S=Q / G$ is the "shape space" capturing the remaining "internal" degrees of freedom. The dynamics can be expressed in the form $\nabla_{\dot{q}} \dot{q}+\operatorname{grad} P=f_{q}{ }^{\sharp}$, where $f_{q} \in F$ and $\nabla$ is the Riemannian (or Levi-Civita) connection.

## A. The Mechanical Connection

We recall that the mechanical connection is given by [14]

$$
\begin{equation*}
H Q=\left\{h_{q} \in T Q:\left\langle\left\langle h_{q}, \xi_{Q}\right\rangle\right\rangle=0 \text { for all } \xi \in G\right\} \tag{2}
\end{equation*}
$$

where the vector field $\xi_{Q}: Q \rightarrow T Q,\left.q \mapsto \frac{d}{d t}\left(\Phi_{\exp t \xi} q\right)\right|_{t=0}$ is the infinitesimal generator of $\xi \in \mathfrak{g}$. We can also describe the mechanical connection using a $\mathfrak{g}$-valued one-form

$$
\begin{equation*}
\mathcal{A}: T Q \rightarrow \mathfrak{g}, v_{q} \mapsto \mathcal{I}(q)^{-1} \mathcal{J}\left(v_{q}\right) \tag{3}
\end{equation*}
$$

where the momentum map $\mathcal{J}: T Q \rightarrow \mathfrak{g}^{*}$ and the locked inertia tensor $\mathcal{I}(q): \mathfrak{g} \rightarrow \mathfrak{g}^{*}$ are defined by

$$
\begin{equation*}
\left\langle\mathcal{J}\left(v_{q}\right) ; \xi\right\rangle=\left\langle\left\langle v_{q}, \xi_{Q}\right\rangle\right\rangle, \quad\langle\mathcal{I}(q) \eta ; \xi\rangle=\left\langle\left\langle\eta_{Q}, \xi_{Q}\right\rangle\right\rangle \tag{4}
\end{equation*}
$$

where $\langle\cdot ; \cdot\rangle$ denotes the natural pairing of covectors and vectors. It can be verified that $\operatorname{ker} \mathcal{A}=H Q$ and

$$
\begin{equation*}
\langle\mathcal{I}(g \cdot q) \xi ; \eta\rangle=\left\langle\mathcal{I}(q) \operatorname{Ad}_{g^{-1}} \xi ; \operatorname{Ad}_{g^{-1}} \eta\right\rangle \tag{5}
\end{equation*}
$$

In any trivialization $q=(s, g)$, any connection can be written

$$
\begin{equation*}
\mathcal{A}=\operatorname{Ad}_{g}\left(g^{-1} d g+\mathbb{A}(s) d s\right) \tag{6}
\end{equation*}
$$

where $\mathbb{A}: T S \rightarrow \mathfrak{g}$ is the local form of the connection, and the local locked inertia tensor is $\mathbb{I}(s)=\mathcal{I}(s, e): \mathfrak{g} \rightarrow \mathfrak{g}^{*}$.

## B. Trivializations, Sections, and Geometric Flat Outputs

Any map $y: Q \rightarrow G$ which is equivariant i.e. $y \circ \Phi_{g}(q)=$ $g y(q)$ induces a trivialization, i.e. a diffeomorphism

$$
\begin{equation*}
\psi: Q \rightarrow S \times G, q \mapsto(\pi(q), y(q)) \tag{7}
\end{equation*}
$$

A section of the bundle is any map belonging to the set

$$
\begin{equation*}
\Gamma(Q)=\{\sigma: S \rightarrow Q: \pi \circ \sigma=\mathrm{id}, \sigma \text { is smooth }\} . \tag{8}
\end{equation*}
$$

Relative to a fixed trivialization, any section can be given as

$$
\begin{equation*}
\sigma: S \rightarrow S \times G, s \mapsto(s, \phi(s)) \tag{9}
\end{equation*}
$$

using a transition map $\phi: S \rightarrow G$. It is well-known that sections and trivializations are in bijective correspondence [15, Prop 9.24], and the group component of the trivialization induced by $\sigma$ is given (relative to the fixed trivialization) by

$$
\begin{equation*}
y: S \times G \rightarrow G,(s, g) \mapsto g \phi(s) \tag{10}
\end{equation*}
$$

A geometric flat output is an equivariant map $y: Q \rightarrow G$ that is also a flat output of the system [12], i.e. for a smooth curve in the flat output space $G$, there is a locally unique dynamically feasible lift in the configuration manifold. ${ }^{1}$

Let the unactuated subbundle be given by $U Q=$ coann $F$. Similar to definitions in [9] and [16], the underactuation distribution can then be defined (as it was in [12]) by

$$
\begin{equation*}
\Delta=\operatorname{span}\left\{X, \nabla_{Y} X: X \in \Gamma(U Q), Y \in \Gamma(T Q)\right\} \tag{11}
\end{equation*}
$$

which is equivariant for any system with broken symmetry. Moreover, when $\operatorname{dim} G=\operatorname{rank} F$ and mild assumptions hold, any solution to the constraint satisfaction problem

$$
\begin{array}{lc}
\text { find } & \sigma \in \Gamma(Q) \\
\text { s.t. } & \Delta \perp \sigma(S) \subset Q \tag{12b}
\end{array}
$$

induces a trivialization whose group component (10) is a geometric flat output [12]. Above, (12b) enforces that $\Delta$ is everywhere orthogonal (with respect to the metric) to the image of $\sigma$ (as a submanifold of $Q$ ). However, a systematic means of obtaining such a section (if it exists) in closed form is not known in the general case.

## III. A Geometric Optimization Problem

In this section, we formulate the search for a geometric flat output in terms of the following optimization problem:

$$
\begin{array}{clr}
\min _{\sigma \in \Gamma(Q)} & J(\sigma) \quad \text { (momentum perturbation cost) } \\
\text { s.t. } & C(\sigma)=0 \quad \text { (orthogonality to } \Delta) \tag{13b}
\end{array}
$$

[^0]where the cost and constraint functionals $J, C: \Gamma(Q) \rightarrow \mathbb{R}$ will be constructed below, and the constraint functional will simply enforce (12b). Since in general we may have $\operatorname{rank} \Delta<\operatorname{dim} G$, solutions to (12a)-(12b) may not be unique, even up to the group action. We therefore propose an additional cost, which penalizes the deviation of the final geometric flat output from the "minimum perturbation coordinates" of the system [17], a trivialization which approximates e.g. an "angular center of mass" when $G=S O(3)$ [18]. This cost will thus encourage the selection of a geometric flat output whose variation most completely captures the entire system's combined momentum, convenient for planning.

## A. Formulating the Cost Functional

We first define an $\mathbb{R}$-valued bilinear form on $Q$ given by

$$
\begin{equation*}
\rho_{\mathcal{A}}:\left(u_{q}, v_{q}\right) \mapsto\left\langle\mathcal{I}(q) \mathcal{A}\left(v_{q}\right) ; \mathcal{A}\left(v_{q}\right)\right\rangle \tag{14}
\end{equation*}
$$

which is easily seen to be symmetric and positive semidefinite. Examining the pullback of $\rho_{\mathcal{A}}$ by $\sigma$ given by

$$
\begin{equation*}
\sigma^{\star} \rho_{\mathcal{A}}:\left(u_{s}, v_{s}\right) \mapsto\left\langle\mathcal{I}(q) \mathcal{A} \circ d \sigma_{s}\left(u_{s}\right) ; \mathcal{A} \circ d \sigma_{s}\left(v_{s}\right)\right\rangle \tag{15}
\end{equation*}
$$

we see that this is also a positive semidefinite symmetric bilinear form, but this time on $S$. Moreover, it will be identically zero over $S$ if and only if $H Q=\operatorname{ker} \mathcal{A}$ is tangent to $\sigma$, which can occur only if the mechanical connection is flat [19] and can be completely integrated to yield a trivialization that amounts to "coordinates" with no "perturbation" whatsoever.

Recall that the trace of a symmetric bilinear form $\omega$ on $S$ is given $\mathrm{by}^{2}(\operatorname{tr} \omega)(s)=\omega\left(v_{s}{ }^{k}, v_{s}{ }^{k}\right)$ for any orthonormal ${ }^{3}$ basis $T_{s} S=\operatorname{span}\left\{v_{s}{ }^{1}, \ldots, v_{s}{ }^{n}\right\}$, and thus the trace of a positive semidefinite form is a non-negative function which vanishes exactly at those points where the form itself vanishes. Hence, $\omega$ will vanish identically over $S$ if and only if the integral of $\operatorname{tr} \omega$ over $S$ vanishes.

Therefore, we define the cost functional

$$
\begin{equation*}
J: \Gamma(Q) \rightarrow \mathbb{R}, \sigma \mapsto \int_{S} \operatorname{tr}\left(\sigma^{\star} \rho_{\mathcal{A}}\right) \operatorname{vol}_{S} \tag{16}
\end{equation*}
$$

where $\operatorname{vol}_{S}$ is the Riemannian volume form [15, Prop. 6.4]. The previous integrand can also be given more explicitly by

$$
\begin{align*}
& \operatorname{tr}\left(\sigma^{\star} \rho_{\mathcal{A}}\right)(s)=\left\langle\mathbb{I}(s)\left(\phi(s)^{-1} d \phi_{s}+\mathbb{A}(s)\right)\left(v_{s}^{k}\right)\right. \\
&\left.\left(\phi(s)^{-1} d \phi_{s}+\mathbb{A}(s)\right)\left(v_{s}^{k}\right)\right\rangle \tag{17}
\end{align*}
$$

which follows from (9) as well as (6) relating the connection form $\mathcal{A}$ and the local form $\mathbb{A}$ and (5) describing the symmetry properties of the locked inertia tensor.

Finally, note that although we consider minimum perturbation coordinates with respect to the mechanical connection (for the reasons already stated), the foregoing analysis is valid for any other principal connection, including the canonical flat connection $(\mathbb{A}=0)$ induced by some other trivialization. Such a choice would penalize (in a differential sense) the deviation of any geometric flat output obtained relative to e.g. some particular body-fixed frame.

[^1]
## B. Reformulating the Orthogonality Constraint

In the original constraint satisfaction problem, the orthogonality constraint (12b) must be enforced pointwise over all of $\sigma(S) \subset Q$. The problem will be much more amenable to practical solution if we express this constraint using a constraint functional that vanishes exactly when the constraint holds. To this end, we define a bilinear form

$$
\begin{equation*}
\rho_{\Delta}:\left(u_{q}, v_{q}\right) \mapsto\left\langle\left\langle\operatorname{proj}_{\Delta}\left(u_{q}\right), \operatorname{proj}_{\Delta}\left(v_{q}\right)\right\rangle\right\rangle \tag{18}
\end{equation*}
$$

where $\operatorname{proj}_{\Delta}: T Q \rightarrow T Q$ is the orthogonal projection onto $\Delta . \rho_{\Delta}$ is clearly symmetric and positive semidefinite, and

$$
\begin{equation*}
\Delta \perp \sigma(S) \Longleftrightarrow \sigma^{\star} \rho_{\Delta}=0 \tag{19}
\end{equation*}
$$

Thus, following a line of reasoning similar to that employed in formulating the cost functional, we enforce the orthogonality constraint using the constraint functional

$$
\begin{equation*}
C: \Gamma(Q) \rightarrow \mathbb{R}, \sigma \mapsto \int_{S} \operatorname{tr}\left(\sigma^{\star} \rho_{\Delta}\right) \operatorname{vol}_{S} \tag{20}
\end{equation*}
$$

where (12b) holds if and only if $C(\sigma)=0$ i.e. (13b) holds.
To make the computation of the integrand more explicit, let $\Delta=\operatorname{span}\left\{\Delta_{1}, \ldots, \Delta_{k}\right\}$ locally where $\left\langle\left\langle\Delta_{i}, \Delta_{j}\right\rangle\right\rangle=\delta_{i j}$ and each $\Delta_{i}$ is equivariant. We then have

$$
\begin{equation*}
\operatorname{proj}_{\Delta}\left(v_{q}\right)=\left\langle\left\langle\Delta_{i}(q), v_{q}\right\rangle\right\rangle \Delta_{i}(q), \tag{21}
\end{equation*}
$$

and moreover

$$
\begin{equation*}
\rho_{\Delta}\left(u_{q}, v_{q}\right)=\left\langle\left\langle u_{q}, \Delta_{i}(q)\right\rangle\right\rangle\left\langle\left\langle\Delta_{i}(q), v_{q}\right\rangle\right\rangle . \tag{22}
\end{equation*}
$$

Since $\Delta_{i}$ is equivariant, there exist maps $\eta_{i}: S \rightarrow \mathfrak{g}$ and vector fields $X_{i}: S \rightarrow T S$ so that in a trivialization, we have

$$
\begin{equation*}
\Delta_{i}:(s, g) \mapsto\left(X_{i}(s), g \eta_{i}(s)\right) \tag{23}
\end{equation*}
$$

Using this symmetry, we may also express the metric as [20]

$$
\begin{align*}
& \left\langle\left\langle\left(u_{s}, u_{g}\right),\left(v_{s}, v_{g}\right)\right\rangle\right\rangle \\
& \quad=\left[\begin{array}{c}
u_{s} \\
g^{-1} u_{g}
\end{array}\right]^{\mathrm{T}}\left[\begin{array}{cc}
m(s) & \mathbb{A}(s)^{\mathrm{T}} \mathbb{I}(s) \\
\mathbb{I}(s) \mathbb{A}(s) & \mathbb{I}(s)
\end{array}\right]\left[\begin{array}{c}
v_{s} \\
g^{-1} v_{g}
\end{array}\right] \tag{24}
\end{align*}
$$

where $u_{q}=\left(u_{s}, u_{g}\right), v_{q}=\left(v_{s}, v_{g}\right) \in T S \times T G$. With this fact in mind, a straightforward computation yields

$$
\begin{align*}
& \operatorname{tr}\left(\sigma^{\star} \rho_{\Delta}\right)(s) \\
& =\left[\begin{array}{c}
v_{s}^{k} \\
\phi(s)^{-1} d \phi_{s}\left(v_{s}^{k}\right)
\end{array}\right] \mathbb{P}_{i}(s)^{\mathrm{T}} \mathbb{P}_{i}(s)\left[\begin{array}{c}
v_{s}^{k} \\
\phi(s)^{-1} d \phi_{s}\left(v_{s}^{k}\right)
\end{array}\right] \tag{25}
\end{align*}
$$

where we define

$$
\mathbb{P}_{i}(s)=\left[\begin{array}{c}
X_{i}(s)  \tag{26}\\
\eta_{i}(s)
\end{array}\right]^{\mathrm{T}}\left[\begin{array}{cc}
m(s) & \mathbb{A}(s)^{\mathrm{T}} \mathbb{I}(s) \\
\mathbb{I}(s) \mathbb{A}(s) & \mathbb{I}(s)
\end{array}\right] .
$$

## C. Gauge Freedom

To study the symmetry of the proposed cost and constraint, consider an inherited $G$-action on $\Gamma(Q)$ given by

$$
\begin{equation*}
\Psi_{g}: \Gamma(Q) \rightarrow \Gamma(Q), \sigma \mapsto\left(s \mapsto \Phi_{g} \circ \sigma(s)\right) \tag{27}
\end{equation*}
$$

In a trivialization, this action is described by

$$
\begin{equation*}
\Psi_{g} \circ(s \mapsto(s, \phi(s)))=s \mapsto(s, g \phi(s)) \tag{28}
\end{equation*}
$$

and moreover we may compute

$$
\begin{align*}
(g \phi(s))^{-1} d(g \phi)_{s}\left(v_{s}\right) & =\phi(s)^{-1} g^{-1} g d \phi_{s}\left(v_{s}\right)  \tag{29}\\
& =\phi(s)^{-1} d \phi_{s}\left(v_{s}\right) \tag{30}
\end{align*}
$$

Thus, in view of the fact that the only dependence of (17) and (25) on $\phi$ (and therefore, $\sigma$ ) takes the previous form, $J$ and $C$ are invariant under $\Psi_{g}$. Therefore, if $\sigma$ is a solution to the problem (13a)-(12b), so is $\Psi_{g}(\sigma)$ for all $g \in G$. Thus, either the addition of a symmetry-breaking cost or constraint can resolve this ambiguity, or this gauge freedom can be freely adjusted after solving the problem once, without re-solving.

## IV. Finite Element Methods

We now transcribe our abstract optimization problem to a finite dimension. Inspired by [13], we parameterize the candidate section (9) using a transition map of the form

$$
\begin{equation*}
\phi: s \mapsto \exp w_{i} \zeta_{i}(s), \tag{31}
\end{equation*}
$$

where each $w_{i} \in \mathbb{R}$ is a weight to be optimized and $\zeta_{i}: S \rightarrow \mathfrak{g}$ is a basis function corresponding to a multivariate polynomial spline over a hypercube mesh (see [13] for a related formulation with piecewise linear elements and a simplex mesh). Basis functions with high-order continuity are desirable, since e.g. two derivatives are necessary to express the group component of the system's acceleration. ${ }^{4}$

## A. First Order Approximation of the Exponential Map

For simplicity, we make the approximation $\exp \xi \approx e+\xi$ for small $\xi$ (which is exact when $G$ is Abelian), so that

$$
\begin{align*}
\phi(s) & \approx e+w_{i} \zeta_{i}(s)  \tag{32}\\
\phi(s)^{-1} d \phi_{s}\left(v_{s}\right) & \approx\left(e-w_{i} \zeta_{i}(s)\right)\left(w_{i} d \zeta_{i}\left(v_{s}\right)\right)  \tag{33}\\
& \approx w_{i} d \zeta_{i}\left(v_{s}\right) \tag{34}
\end{align*}
$$

The cost functional integrand (17) is thus approximated by

$$
\begin{align*}
& \operatorname{tr}\left(\sigma^{\star} \rho_{\mathcal{A}}\right)(s) \\
& \quad \approx\left\langle\mathbb{I}(s)\left(w_{i} d \zeta_{i}+\mathbb{A}\right)\left(v_{s}^{k}\right) ;\left(w_{j} d \zeta_{j}+\mathbb{A}\right)\left(v_{s}^{k}\right)\right\rangle \tag{35}
\end{align*}
$$

and similarly for the constraint functional integrand (25),

$$
\begin{align*}
& \operatorname{tr}\left(\sigma^{\star} \rho_{\Delta}\right)(s) \\
& \quad \approx\left[\begin{array}{c}
v_{s}^{k} \\
w_{h} d \zeta_{h}\left(v_{s}{ }^{k}\right)
\end{array}\right]^{\mathrm{T}} \mathbb{P}_{i}(s)^{\mathrm{T}} \mathbb{P}_{i}(s)\left[\begin{array}{c}
v_{s}^{k} \\
w_{j} d \zeta_{j}\left(v_{s}^{k}\right)
\end{array}\right] \tag{36}
\end{align*}
$$

## B. Gaussian Quadrature Approximation of the Integrals

We employ Gaussian quadrature [22] to approximate the integrals in the functionals. Considering coordinates $s^{i}$ for $S=\bigcup_{j=1}^{m} E_{j}$ where $E_{j}$ are finitely many disjoint elements,

$$
\begin{align*}
\int_{S} f(s) \operatorname{vol}_{S} & =\sum_{j=1}^{m} \int_{E_{j}}(f(s) \sqrt{\operatorname{det} \mathbb{G}(s)}) d s^{1} \ldots d s^{n}  \tag{37}\\
& \approx \sum_{j=1}^{m} \sum_{i=1}^{p} c_{i j}\left(f\left(s_{i j}\right) \sqrt{\operatorname{det} \mathbb{G}\left(s_{i j}\right)}\right), \tag{38}
\end{align*}
$$

[^2]

Fig. 1. For the Planar Rocket [12, Ex. 1], we depict the finite element boundaries (i.e. the knot points of the quintic spline) and the restriction of $\Delta$ to $\sigma\left(\mathbb{S}^{1}\right) \subset S E(2)$, which can be seen to be orthogonal to the solution $\sigma: \mathbb{S}^{1} \rightarrow S E(2)$ to (41a)-(41c). (Technically, we plot $\Delta^{b} \subset T^{*} Q$, and vectors and covectors appearing at right angles amounts to orthogonality.)
where each $c_{i j} \in \mathbb{R}$ and $s_{i j} \in S$ is a predetermined optimal weight and sampling point respectively, and $\mathbb{G}(s)$ is the induced Riemannian metric in the coordinates $s^{i}$. Since Gaussian quadrature with $p$ samples is exact when the integrand is a polynomial of degree $2 p-1$ or less, we expect this to be a good approximation if our mesh is sufficiently fine and $\zeta_{i}$ is a polynomial of degree $p$ or less in $s^{i}$.

## C. Approximate Transcription to a Quadratic Program

It is clear that pointwise over $S$, both (35) and (36) are quadratic forms in the weights $w_{i}$. Since quadrature approximates integrals via a weighted sum, our final approximations of (16) and (20) can be expressed in the form

$$
\begin{align*}
J(\sigma) & \approx \frac{1}{2} w_{i} A_{i j} w_{j}+b_{i} w_{i}+c  \tag{39}\\
C(\sigma) & \approx \frac{1}{2} w_{i} D_{i j} w_{j}+e_{i} w_{i}+f=0 \tag{40}
\end{align*}
$$

where $A_{i j}$ and $D_{i j}$ are positive semidefinite. Due to transcription, (40) may be infeasible even if a geometric flat output exists. However, because (40) is nonnegative in view of (36), we relax the orthogonality constraint to simply enforce minimization of $C(\sigma)$ instead, where minimization to zero (when feasible) corresponds to exact constraint satisfaction.
Thus, we obtain an approximate transcription of the continuum problem (13a)-(13b) to a Quadratic Program (QP):

$$
\begin{array}{rlll}
\min _{w \in \mathbb{R}^{N}} & \frac{1}{2} w_{i} A_{i j} w_{j}+b_{i} w_{i}+c & & \text { (momentum cost) } \\
\text { s.t. } & D_{i j} w_{j}+e_{i} & =0 & \\
& w_{i} \zeta_{i}\left(s_{0}\right) & =\log \left(g_{0}\right) &  \tag{41c}\\
\text { (breaogs syality }) \\
\end{array}
$$

where (41c) is added to resolve the gauge freedom. The QP is sparse due to the choice of basis functions with local support, permitting efficient solution even for very fine meshes.

## V. Results

We implement the proposed method to identify geometric flat outputs on two benchmark systems, namely the Planar Rocket [12, Ex. 1] on $S E(2)$ (an $\mathbb{R}^{2}$ bundle over $\mathbb{S}^{1}$ ) and the Planar Aerial Manipulator [12, Ex. 2] on $S E(2) \times \mathbb{S}^{1}$ (an $S E(2)$ bundle over $\mathbb{S}^{1}$ ). In both examples, we employ $C^{2}$ basis functions corresponding to quintic Hermite splines


Fig. 2. Visualization of the numerical geometric flat output obtained for the Planar Aerial Manipulator [12, Ex. 2], which amounts to a frame parallel to the end effector with a $\phi$-dependent offset. When $\ell_{q}=0$, the offset goes to zero. Physical parameters have been chosen to exaggerate the offset for clarity. The frame in red corresponds to the angle $\phi$ visualized above.
over an evenly divided mesh on $\mathbb{S}^{1}$ with $m=8$ elements, and perform Gaussian quadrature with $p=6$ sample points per element. The orthogonal section obtained is visualized in Fig. 1 for the Planar Rocket, meanwhile the final geometric flat output is visualized in Fig. 2 for the Planar Aerial Manipulator. With suitable (and easily chosen) gauge freedom constraints, the numerical geometric flat outputs identified for both systems show precise numerical agreement with the closed form geometric flat outputs previously given in [12].

## VI. Conclusion

In this work, we have used Riemannian geometry and Lie group theory to cast the search for a geometric flat output as an optimization problem. Explicit cost and constraint functionals are formulated, which are shown to be naturally invariant with respect to the system's symmetry. We perform an approximate transcription of the continuum problem to a finite dimensional quadratic program using finite element methods, Gaussian quadrature, and a small angle approximation. The approach is implemented for two example systems, and close agreement with closed-form solutions provides evidence in favor of the method's feasibility for accurate identification of geometric flat outputs for more complex systems, for which closed form solutions are unknown.
In our ongoing work, we will pursue an iterative approach, where at each iteration, the exponential map is linearized about the current guess until convergence. Since this process can be performed offline, the computational requirements should not be an obstacle. In order to apply the approach to more complex systems, we will explore computing the relevant quantities (e.g. the underactuation distribution $\Delta$ and the mechanical connection $\mathcal{A}$ ) numerically from a standard modeling format (e.g. a URDF [23]), adapting efficient multibody dynamics solvers (e.g. Pinocchio [24]) to perform the necessary covariant differentations by using a suitable factorization of the Coriolis forces [25]. We will also explore the extent to which the relaxed orthogonality constraint that simply minimizes $C(\sigma)$ may yield an "almost flat output" when a truly orthogonal section does not exist, even in the continuum setting. Finally, our approach amounts to a continuous deformation of an existing section; thus, a potential pitfall is that in general, all sections of a principal bundle need not be homotopic, although over a contractible shape space (e.g. a local region), all sections are homotopic.

## References

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[^0]:    ${ }^{1}$ A geometric flat output may also be induced by a local (e.g. almost global) section; for brevity, here we consider trivial bundles and global geometric flat outputs, but direct the reader to [12] for a broader discussion.

[^1]:    ${ }^{2}$ For all index notation, we follow the Einstein summation convention.
    ${ }^{3}$ A $G$-invariant metric on $Q$ induces a natural metric on $S=Q / G$ given by $\left(u_{s}, v_{s}\right) \mapsto\left\langle\left\langle\left(u_{s}\right)_{q}^{H Q},\left(v_{s}\right)_{q}^{H Q}\right\rangle\right\rangle$ for any $q \in \pi^{-1}(s)$, where $(\cdot)_{q}^{H Q}: T_{s} S \rightarrow T_{q} Q$ is the horizontal lift via the mechanical connection.

[^2]:    ${ }^{4}$ Note that Hermite polynomial basis functions with $C^{2}$ smoothness have been given up to at least three dimensions [21], but may be available in higher dimensions as well. Since for our purposes, $\operatorname{dim} S=\operatorname{rank} U Q$, we are concerned with low-dimensional cases, an advantage of approximating the section $\sigma: S \rightarrow Q$ instead of the flat output $y: Q \rightarrow G$ directly.

